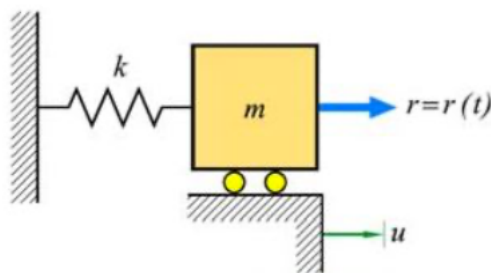




Assignment 10- Dynamics

Question 1)

In the dynamic system of slide 6, let  $r(t)$  be a constant force  $F$ . What is the effect of  $F$  on the time-dependent displacement  $u(t)$  and the natural frequency of vibration of the system?



I have to consider the second law of Newton about forces balance, that is:

$$F = m * a$$

In this case the force is constant and if we consider  $u(t)$  the displacement the equation will be:

$$F = m\ddot{u} + ku$$

Now I obtained the solution for a non-homogeneous ordinary differential equation ODE, which include a general solution and a particular one.

$$0 = m\ddot{u} + ku$$

General solution:

$$u(t) = A\sin(\omega t + \phi) + B\cos(\omega t + \phi)$$

Where  $A$  is the amplitude of motion;  $\omega = \sqrt{\frac{k}{m}}$  is the natural frequency of vibration.

Calculation of the solution for the initial condition  $u(t = 0) = 0, \phi = 0$ :

General solution:

$$u(t) = B\cos(\omega t)$$

Particular solution:

$$F = u_p K \rightarrow u_p = \frac{F}{k}$$

Total Equation:

$$u(t) = B\cos(\omega t) + u_p = B\cos(\omega t) + \frac{F}{k}$$

Now I have to find the value of B.

$$\text{initial condition } u(t = 0) = 0 : A = 0 \quad \text{and } B = -\frac{F}{k}$$

So the final force equation will be:

$$u(t) = \frac{F}{k}(1 - \cos(\omega t)) = \frac{F}{k}(1 - \cos(t * \left(\sqrt{\frac{k}{m}}\right)))$$

Observation: F does not produce a phase shift and doesn't affect the natural vibration frequency of the system  $\omega$ . It affects the amplitude of the vibration and the range of movement.

### Question 2)

A weight whose mass is  $m$  is placed at the middle of a uniform axial bar of length  $L$  that is clamped at both ends. The mass of the bar may be neglected. Estimate the natural frequency of vibration in terms of  $m$ ,  $L$ ,  $E$  and  $A$ . Suggestion: First determine the effective  $k$ .

The natural frequency of vibration is  $w = \sqrt{K/m}$ , so I have to find  $k$ . The first equation was:

$$F = ku$$

So  $K$  will be:

$$K = \frac{F}{u}$$

We have to consider the maximum vertical displacement in the middle of the beam (point of the load application). This displacement is:

$$u = \frac{F}{EI} * \frac{l^3}{192}$$

Where:

- $E$  in the Young modulus,
- $F = m * g$
- $I$  the inertia that is equal to  $I = \pi * \frac{d^4}{12} = \frac{A^2}{4\pi}$

The expression of the natural frequency will be:

$$w = \sqrt{K/m} = \frac{4A}{L} * \sqrt{\frac{E}{mL}}$$

### Question 3

Use the expression  $\mathbf{m} = \int \mathbf{N}^T * \mathbf{N} * \rho dV$  to derive the following mass matrix.

It is possible to define N into two shape function, which are:

$$N_1 = 1 - \frac{x}{L}$$
$$N_2 = \frac{x}{L}$$

The density and the cross section are constant, so it is possible to write the consistent element mass matrix as following:

$$M = \rho A \int_0^L \mathbf{N}^T \mathbf{N} dx$$

Where

$$\mathbf{N}^T \mathbf{N} = \begin{bmatrix} N_1^2 & N_1 N_2 \\ N_1 N_2 & N_2^2 \end{bmatrix}$$

And so:

$$M = \rho A \int_0^L \mathbf{N}^T \mathbf{N} dx = \rho A \int_0^L \begin{bmatrix} N_1^2 & N_1 N_2 \\ N_1 N_2 & N_2^2 \end{bmatrix} dx = \frac{\rho A}{L^2} \int_0^L \begin{bmatrix} (l-x)^2 & (l-x)x \\ (l-x)x & x^2 \end{bmatrix} dx =$$

$$M = \rho A \begin{bmatrix} \frac{L}{3} & \frac{L}{6} \\ \frac{L}{6} & \frac{L}{3} \end{bmatrix}$$

Question 4)

Obtain also the mass matrix of a two-node, linear displacement element with a variable cross-sectional area that varies from  $A_1$  to  $A_2$ .

Taking into account the two shape functions  $N_1, N_2$ , of the last question I obtain a linear interpolation for the cross section of linear displacement element:

$$A(x) = A_1 * N_1(x) + A_2 * N_2(x)$$

Now I'm considering the variation of the cross section, so I can calculate the value of M as:

$$M = \rho(A_1 \int_0^L \begin{bmatrix} N_1^3 & N_1^2 N_2 \\ N_1^2 N_2 & N_1 N_2^2 \end{bmatrix} dx + A_2 \int_0^L \begin{bmatrix} N_1^2 N_2 & N_1 N_2^2 \\ N_1 N_2^2 & N_2^3 \end{bmatrix} dx)$$

Computing the integral I obtained:

$$\int_0^L N_1^3 dx = \frac{L}{4}$$

$$\int_0^L N_2^3 dx = \frac{L}{4}$$

$$\int_0^L N_1^2 N_2 dx = \frac{L}{12}$$

$$\int_0^L N_1 N_2^2 dx = \frac{L}{12}$$

$$M = \rho L (A_1 \begin{bmatrix} \frac{1}{4} & \frac{1}{12} \\ \frac{1}{12} & \frac{1}{12} \end{bmatrix} + A_2 \begin{bmatrix} \frac{1}{12} & \frac{1}{12} \\ \frac{1}{12} & \frac{1}{4} \end{bmatrix})$$

Question 5)

*A uniform two-node bar element is allowed to move in a 3D space. The nodes have only translational d.o.f. What is the diagonal mass matrix of the element?*

The element bar has 6 degrees of freedom ( 3 displacement of each node) and the mass matrix correspond to a two-nodal bar element is 6x6. The mass matrix has only term on diagonal if the nodes can only have translational degree of freedom:

$$M = \frac{LA\rho}{2} \begin{bmatrix} 1 & 0 & 0 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 \end{bmatrix}$$